

AA242B: MECHANICAL VIBRATIONS

Analytical Dynamics of Discrete Systems

These slides are based on the recommended textbook: M. Géradin and D. Rixen, "Mechanical Vibrations: Theory and Applications to Structural Dynamics," Second Edition, Wiley, John & Sons, Incorporated, ISBN-13:9780471975465

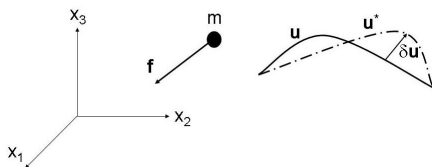


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- 2 Principle of Virtual Work for a System of N Particles
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- 4 Lagrange Equations in the General Case



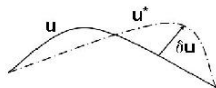
Principle of Virtual Work for a Particle



- Particle mass m
- Particle force
 - force vector $\mathbf{f} = [f_1 \quad f_2 \quad f_3]^T$
 - force component $f_i, i = 1, \dots, 3$
- Particle displacement
 - displacement vector $\mathbf{u} = [u_1 \quad u_2 \quad u_3]^T$
 - displacement component $u_i, i = 1, \dots, 3$
 - motion trajectory $\mathbf{u}(t)$ where t denotes time



Principle of Virtual Work for a Particle



- Particle virtual displacement
 - arbitrary displacement \mathbf{u}^* (can be zero)
 - virtual displacement $\delta \mathbf{u} = \mathbf{u}^* - \mathbf{u} \Rightarrow$ arbitrary by definition
 - family of arbitrary virtual displacements defined in a time-interval $[t_1, t_2]$ and satisfying the variational constraints

$$\delta \mathbf{u}(t_1) = \delta \mathbf{u}(t_2) = \mathbf{0}$$

- Important property

$$\frac{d}{dt}(\delta u_i) = \frac{d}{dt}(u_i^* - u_i) = \frac{du_i^*}{dt} - \frac{du_i}{dt} = \dot{u}_i^* - \dot{u}_i = \delta \dot{u}_i$$

$$\Rightarrow \frac{d}{dt}(\delta) = \delta\left(\frac{d}{dt}\right) \quad (\text{commutativity})$$



└ Principle of Virtual Work for a Particle

■ Equilibrium

- strong form

$$m\ddot{\mathbf{u}} - \mathbf{f} = \mathbf{0} \Rightarrow m\ddot{u}_i - f_i = 0, \quad i = 1, \dots, 3$$

- weak form

$$\forall \delta \mathbf{u}, \quad (\delta \mathbf{u}^T)(m\ddot{\mathbf{u}} - \mathbf{f}) = 0 \Rightarrow \sum_{i=1}^3 (m\ddot{u}_i - f_i) \delta u_i = 0$$

$$\Rightarrow (m\ddot{u}_i - f_i) \delta u_i = 0, \quad i = 1, \dots, 3$$

- $\delta \mathbf{u}^T (m\ddot{\mathbf{u}} - \mathbf{f}) = (m\ddot{\mathbf{u}} - \mathbf{f})^T \delta \mathbf{u}$ is homogeneous to a work
 \Rightarrow virtual work (δW)
- Virtual work principle

The virtual work produced by the effective forces acting on a particle during a virtual displacement is equal to zero



└ Principle of Virtual Work for a System of N Particles

- N particles: $k = 1, \dots, N$
- Equilibrium

$$m\ddot{\mathbf{u}}_k - \mathbf{f}_k = \mathbf{0}, \quad k = 1, \dots, N$$

- Family of virtual displacements $\delta\mathbf{u}_k = \mathbf{u}_k^* - \mathbf{u}_k$ satisfying the variational constraints

$$\delta\mathbf{u}_k(t_1) = \delta\mathbf{u}_k(t_2) = \mathbf{0} \quad (1)$$

- Virtual work

$$m\ddot{\mathbf{u}}_k - \mathbf{f}_k = \mathbf{0} \Rightarrow \sum_{k=1}^N \delta\mathbf{u}_k^T (m\ddot{\mathbf{u}}_k - \mathbf{f}_k) = \sum_{k=1}^N (m\ddot{\mathbf{u}}_k - \mathbf{f}_k)^T \delta\mathbf{u}_k = 0$$



Principle of Virtual Work for a System of N Particles

- Conversely, $\forall \delta \mathbf{u}_k$ compatible with the variational constraints (1)

$$\sum_{k=1}^N \delta \mathbf{u}_k^T (m \ddot{\mathbf{u}}_k - \mathbf{f}_k) = 0 \Rightarrow \sum_{k=1}^N \sum_{i=1}^3 (m_k \ddot{u}_{i_k} - f_{i_k}) \delta u_{i_k} = 0 \quad (2)$$

- If (2) is true $\forall \delta \mathbf{u}_k$ compatible with (1) \Rightarrow (2) is true for $\delta \mathbf{u}_j = [1 \ 0 \ 0]^T$ ($\delta \mathbf{u}_k = [0 \ 0 \ 0]^T$ for $k \neq j$), $\delta \mathbf{u}_j = [0 \ 1 \ 0]^T$ ($\delta \mathbf{u}_k = [0 \ 0 \ 0]^T$ for $k \neq j$), and $\delta \mathbf{u}_j = [0 \ 0 \ 1]^T$ ($\delta \mathbf{u}_k = [0 \ 0 \ 0]^T$ for $k \neq j$), $t \in]t_1, t_2[$

$$\Rightarrow m_j \ddot{u}_{ij} - f_{ij} = 0, \quad i = 1, \dots, 3, \quad j = 1, \dots, N$$

$$\left(\Rightarrow \sum_{k=1}^N m_k \ddot{u}_{i_k} - f_{i_k} = 0, \quad i = 1, \dots, 3 \right)$$

If the virtual work equation is satisfied for any displacement compatible with the variational constraints, the system (of particles) is in dynamic equilibrium



- Major result

dynamic equilibrium \Leftrightarrow virtual work principle



└ Principle of Virtual Work for a System of N Particles

└ Kinematic Constraints

- In the absence of (kinematic) constraints, the state of a system of N particles can be defined by $3N$ displacement components

$$u_{i_k}, \quad i = 1, \dots, 3, \quad k = 1, \dots, N$$

- Instantaneous configuration

$$\xi_{i_k} = x_{i_k} + u_{i_k}(\mathbf{x}, t) \Rightarrow 3N \text{ dofs}$$

- However, most mechanical systems incorporate some sort of constraints
 - holonomic constraints
 - non-holonomic constraints



- Principle of Virtual Work for a System of N Particles

- Kinematic Constraints

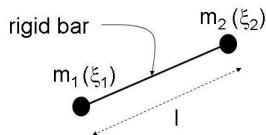
- Holonomic constraints

- two types

- rheonomic constraints: defined by $c(\xi_{i_k}, t) = 0$ (no explicit dependence on any velocity)
- scleronomic constraints: defined by $c(\xi_{i_k}) = 0$ (no explicit dependence on any velocity or time)

- a holonomic constraint reduces by 1 the number of dofs of a mechanical system

- example



- rigidity \Rightarrow conservation of length $\Rightarrow \sum_{i=1}^3 (\xi_{i_2} - \xi_{i_1})^2 = l^2$

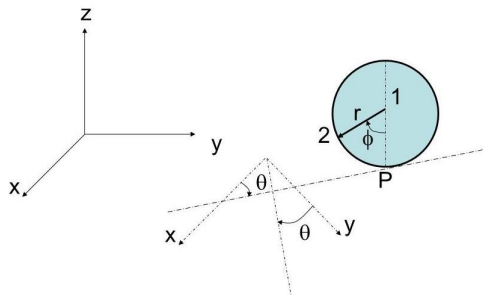


- Principle of Virtual Work for a System of N Particles

- Kinematic Constraints

- Non-holonomic constraints

- defined by $c(\dot{\xi}_{i_k}, \xi_{i_k}, t) = 0$
- example



- no slip \Rightarrow speed of point $P = 0$

$$\Rightarrow \begin{cases} \dot{x}_1 = 0 - r\dot{\phi} \cos \theta & \Rightarrow \dot{x}_1 + r\dot{\phi} \cos \theta = 0 \\ \dot{y}_1 = 0 + r\dot{\phi} \sin \theta & \Rightarrow \dot{y}_1 - r\dot{\phi} \sin \theta = 0 \end{cases}$$

- in addition

$$\begin{cases} x_2 - x_1 = r \sin \phi \cos \theta & y_2 - y_1 = -r \sin \phi \sin \theta & z_2 - z_1 = -r \cos \phi \\ z_1 = r \end{cases}$$



└ Principle of Virtual Work for a System of N Particles

└ Kinematic Constraints

- example (continue)
 - hence, this system has
 - 8 variables: $x_1, y_1, z_1, x_2, y_2, z_2, \theta, \phi$
 - 4 holonomic constraints
 - 2 non-holonomic constraints
 - in general, $c(\dot{\xi}_{i_k}, \xi_{i_k}, t) = 0$ is not integrable and therefore non-holonomic constraints do not reduce the number of dofs of a mechanical system
 - therefore, the mechanical system in the above example (wheel) has $8 - 4 = 4$ dofs
 - 2 translations in the rolling plane
 - 2 rotations



└ Principle of Virtual Work for a System of N Particles

└ Generalized Displacements

- Let n denote the number of dofs of a mechanical system: for example, for a system with N material points and R holonomic constraints, $n = 3N - R$
- The generalized coordinates of this system are defined as the n configuration parameters (q_1, q_2, \dots, q_n) in terms of which the displacements can be expressed as

$$u_{ik}(\mathbf{x}, t) = U_{ik}(q_1, q_2, \dots, q_n, t)$$

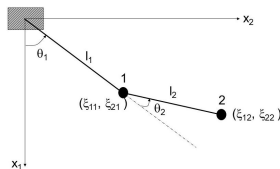
- If the system is not constrained by any non-holonomic constraint, then the generalized coordinates (q_1, q_2, \dots, q_n) are independent: they can vary arbitrarily without violating the kinematic constraints



- Principle of Virtual Work for a System of N Particles

- Generalized Displacements

- Example



- holonomic constraint HC1: $\xi_{11}^2 + \xi_{21}^2 = l_1^2$
- holonomic constraint HC2: $(\xi_{12} - \xi_{11})^2 + (\xi_{22} - \xi_{21})^2 = l_2^2$
 $\implies 4 - 2 = 2\text{dofs}$
- one possible choice of (q_1, q_2) is (θ_1, θ_2)

$$\implies \begin{cases} \xi_{11} = l_1 \cos \theta_1 & \xi_{21} = l_1 \sin \theta_1 \\ \xi_{12} = l_1 \cos \theta_1 + l_2 \cos(\theta_1 + \theta_2) & \xi_{22} = l_1 \sin \theta_1 + l_2 \sin(\theta_1 + \theta_2) \end{cases}$$



- Principle of Virtual Work for a System of N Particles

- Generalized Displacements

- Virtual displacements

$$u_{i_k}(\mathbf{x}, t) = U_{i_k}(q_1, q_2, \dots, q_n, t) \Rightarrow \delta u_{i_k} = \sum_{s=1}^n \frac{\partial U_{i_k}}{\partial q_s} \delta q_s$$

- Virtual work equation

$$\sum_{s=1}^n \left[\sum_{k=1}^N \sum_{i=1}^3 (m_k \ddot{u}_{i_k} - f_{i_k}) \frac{\partial U_{i_k}}{\partial q_s} \right] \delta q_s = 0$$

- Second term in above equation can be written as $\sum_{s=1}^n Q_s \delta q_s$ where

$$Q_s = \sum_{k=1}^N \sum_{i=1}^3 f_{i_k} \frac{\partial U_{i_k}}{\partial q_s}$$

is the generalized force conjugate to q_s



└ Hamilton's Principle for Conservative Systems and Lagrange Equations

- Sir William Rowan Hamilton (4 August 1805 - 2 September 1865)



- Irish physicist, astronomer, and mathematician
- contributions: classical mechanics, optics, and algebra (inventor of quaternions), and most importantly, reformulation of Newtonian mechanics (now called Hamiltonian mechanics)
- impact: modern study of electromagnetism, development of quantum mechanics



└ Hamilton's Principle for Conservative Systems and Lagrange Equations

- Hamilton's principle: $-\int_{t_1}^{t_2}$ virtual work principle = 0!

$$-\int_{t_1}^{t_2} \left[\sum_{k=1}^N \sum_{i=1}^3 (m_k \ddot{u}_{i_k} - f_{i_k}) \delta u_{i_k} \right] dt = 0$$

where δu_{i_k} are arbitrary but compatible with eventual constraints and verify the end conditions (previously referred to as variational constraints)



Hamilton's Principle for Conservative Systems and Lagrange Equations

- First, assume that \mathbf{f}_k derives from a potential $\mathcal{V}(\xi_i)$ — that is, \mathbf{f}_k is a conservative force

$$\Rightarrow \exists \mathcal{V}(\xi_{1_k}, \xi_{2_k}, \xi_{3_k}) / \mathbf{f}_k = -\nabla \mathcal{V}(\xi_{1_k}, \xi_{2_k}, \xi_{3_k})$$

- Virtual work of \mathbf{f}_k

$$\delta W = \sum_{k=1}^N \sum_{i=1}^3 f_{ik} \delta u_{ik} = - \sum_{k=1}^N \sum_{i=1}^3 \frac{\partial \mathcal{V}}{\partial \xi_{ik}} \delta u_{ik} = - \sum_{k=1}^N \sum_{i=1}^3 \frac{\partial \mathcal{V}}{\partial \xi_{ik}} \delta \xi_{ik} = -\delta \mathcal{V}$$

- $\delta W = -\delta \mathcal{V}$ and $\delta W = \sum_{s=1}^n Q_s \delta q_s$

$$\Rightarrow \boxed{Q_s = -\frac{\partial \mathcal{V}}{\partial q_s}}$$

- What about the virtual work of the inertia forces?



└ Hamilton's Principle for Conservative Systems and Lagrange Equations

- Note that

$$\begin{aligned}\frac{d}{dt}(m_k \dot{u}_{ik} \delta u_{ik}) &= m_k \ddot{u}_{ik} \delta u_{ik} + m_k \dot{u}_{ik} \delta \dot{u}_{ik} \\ &= m_k \ddot{u}_{ik} \delta u_{ik} + \delta \left(\frac{1}{2} m_k \dot{u}_{ik}^2 \right) \\ \implies \delta W &= m_k \ddot{u}_{ik} \delta u_{ik} = \frac{d}{dt}(m_k \dot{u}_{ik} \delta u_{ik}) - \delta \left(\frac{1}{2} m_k \dot{u}_{ik}^2 \right)\end{aligned}$$

- The kinetic energy of a system of N particles can be defined as

$$\mathcal{T} = \frac{1}{2} \sum_{k=1}^N \sum_{i=1}^3 m_k \dot{u}_{ik}^2$$



Hamilton's Principle for Conservative Systems and Lagrange Equations

- Hence, Hamilton's principle for a conservative system can be written as

$$\begin{aligned}
 & - \int_{t_1}^{t_2} \left[\sum_{k=1}^N \sum_{i=1}^3 (m_k \ddot{u}_{ik} - f_{ik}) \delta u_{ik} \right] dt = 0 \\
 \Rightarrow & - \int_{t_1}^{t_2} \left[\sum_{k=1}^N \sum_{i=1}^3 \frac{d}{dt} (m_k \dot{u}_{ik} \delta u_{ik}) - \delta \left(\frac{1}{2} m_k \dot{u}_{ik}^2 \right) \right] dt + \int_{t_1}^{t_2} (-\delta \mathcal{V}) dt = 0 \\
 \Rightarrow & - \sum_{k=1}^N \sum_{i=1}^3 m_k \dot{u}_{ik} \delta u_{ik} \Big|_{t_1}^{t_2} + \delta \int_{t_1}^{t_2} (\mathcal{T} - \mathcal{V}) dt = 0
 \end{aligned}$$

- Recall the generalized displacements

$$\begin{aligned}
 \{q_s\} & \Rightarrow u_{ik}(\mathbf{x}, t) = U_{ik}(q_s, t) \Rightarrow \dot{u}_{ik} = h(q_s, \dot{q}_s, t) \\
 & \Rightarrow \xi_{ik} = x_{ik} + u_{ik}(\mathbf{x}, t) = x_{ik} + U_{ik}(q_s, t) = g(q_s, t)
 \end{aligned}$$

$$\Rightarrow \boxed{\mathcal{T} = \mathcal{T}(\mathbf{q}, \dot{\mathbf{q}}, t) \quad \text{and} \quad \mathcal{V} = \mathcal{V}(\mathbf{q}, t)}$$



Hamilton's Principle for Conservative Systems and Lagrange Equations

- Note that $u_{i_k}(\mathbf{x}, t) = U_{i_k}(q_s, t) \Rightarrow \delta u_{i_k} = \sum_{s=1}^n \frac{\partial U_{i_k}}{\partial q_s} \delta q_s$

- Now, recall the end conditions

$$\delta u_{i_k}(t_1) = 0 \Rightarrow \delta q_s(t_1) = 0 \quad \text{and} \quad \delta u_{i_k}(t_2) = 0 \Rightarrow \delta q_s(t_2) = 0$$

- Therefore, Hamilton's principle (HP) can be written as

$$\delta \int_{t_1}^{t_2} [\mathcal{T}(\mathbf{q}, \dot{\mathbf{q}}, t) - \mathcal{V}(\mathbf{q}, t)] dt = 0 \quad \forall \delta \mathbf{q} / \delta \mathbf{q}(t_1) = \delta \mathbf{q}(t_2) = 0$$

where

$$\mathbf{q} = [q_1 \quad q_2 \quad \cdots \quad q_s \quad \cdots \quad q_n]^T$$



Hamilton's Principle for Conservative Systems and Lagrange Equations

Equations of motion

$$\blacksquare \delta \mathcal{T} = \sum_{s=1}^n \left(\frac{\partial \mathcal{T}}{\partial q_s} \delta q_s + \frac{\partial \mathcal{T}}{\partial \dot{q}_s} \delta \dot{q}_s \right)$$

$$\blacksquare \delta \mathcal{V} = - \sum_{s=1}^n Q_s \delta q_s$$

$$\blacksquare \text{HP} \rightarrow \int_{t_1}^{t_2} \left[\sum_{s=1}^n \frac{\partial \mathcal{T}}{\partial \dot{q}_s} \delta \dot{q}_s + \left(\frac{\partial \mathcal{T}}{\partial q_s} + Q_s \right) \delta q_s \right] dt = 0$$

■ integrate by parts and apply the end conditions

$$\begin{aligned} \sum_{s=1}^n \frac{\partial \mathcal{T}}{\partial \dot{q}_s} \delta q_s \Big|_{t_1}^{t_2} &- \int_{t_1}^{t_2} \sum_{s=1}^n \frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) \delta q_s dt \\ &+ \int_{t_1}^{t_2} \sum_{s=1}^n \left(\frac{\partial \mathcal{T}}{\partial q_s} + Q_s \right) \delta q_s dt = 0 \end{aligned}$$

$$\Rightarrow \int_{t_1}^{t_2} \sum_{s=1}^n \left[- \frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}}{\partial q_s} + Q_s \right] \delta q_s dt = 0$$



Hamilton's Principle for Conservative Systems and Lagrange Equations

- The Lagrange equations of motion

$$\int_{t_1}^{t_2} \sum_{s=1}^n \left[-\frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}}{\partial q_s} + Q_s \right] \delta q_s dt = 0$$

$$\forall \delta \mathbf{q} / \delta \mathbf{q}(t_1) = \delta \mathbf{q}(t_2) = 0$$

$$\Rightarrow \boxed{-\frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}}{\partial q_s} + Q_s = 0}$$

- $-\frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}}{\partial q_s}$: generalized inertia forces
- Q_s : generalized internal and external forces



└ Hamilton's Principle for Conservative Systems and Lagrange Equations

- Joseph-Louis (Giuseppe Lodovico), comte de Lagrange (25 January 1736 - 10 April 1813)

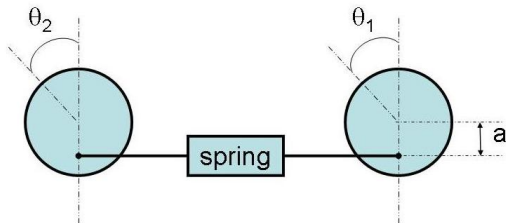


- Italian-born mathematician and astronomer
- contributions: analysis, number theory, and classical and celestial mechanics
- succeeded Euler in 1766 as the director of mathematics at the Prussian Academy of Sciences in Berlin
- impact: his treatise on analytical mechanics written in Berlin and first published in 1788 formed a basis for the development of mathematical physics in the 19th century
- moved to France in 1787 and became a member of the French Academy
- survived the French Revolution and became the first professor of analysis at the École Polytechnique upon its opening in 1794



Hamilton's Principle for Conservative Systems and Lagrange Equations

- Equations of equilibrium $\sum_{k=1}^N m_k \ddot{u}_{ik} - f_{ik} = 0$, $i = 1, \dots, 3$: simple, but can be difficult to formulate analytically for complex systems
- Lagrange's equations of motion $-\frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}}{\partial q_s} + Q_s = 0$: more complex, but can simplify the analytical solution of complex dynamic problems



Hamilton's Principle for Conservative Systems and Lagrange Equations

Classification of Inertia Forces

- Recall $u_{i_k} = U_{i_k}(\mathbf{q}, t) \Rightarrow \dot{u}_{i_k} = \sum_{s=1}^n \frac{\partial U_{i_k}}{\partial q_s} \dot{q}_s + \frac{\partial U_{i_k}}{\partial t}$
- Substitute in the expression of the kinetic energy
 $\Rightarrow \mathcal{T} = \mathcal{T}_0(\mathbf{q}, t) + \mathcal{T}_1(\mathbf{q}, \dot{\mathbf{q}}, t) + \mathcal{T}_2(\mathbf{q}, \dot{\mathbf{q}}, t)$ where \mathcal{T}_0 , \mathcal{T}_1 , and \mathcal{T}_2 are homogeneous forms¹ of degree 0, 1, and 2 in \dot{q}_s , respectively, and are given by

$$\mathcal{T}_0(\mathbf{q}, t) = \frac{1}{2} \sum_{k=1}^N \sum_{i=1}^3 m_k \left(\frac{\partial U_{i_k}}{\partial t} \right)^2 \quad (\text{transport kinetic energy})$$

$$\mathcal{T}_1(\mathbf{q}, \dot{\mathbf{q}}, t) = \sum_{s=1}^n \sum_{k=1}^N \sum_{i=1}^3 \frac{\partial U_{i_k}}{\partial t} m_k \frac{\partial U_{i_k}}{\partial q_s} \dot{q}_s \quad (\text{mutual kinetic energy})$$

$$\mathcal{T}_2(\mathbf{q}, \dot{\mathbf{q}}, t) = \frac{1}{2} \sum_{s=1}^n \sum_{r=1}^n \sum_{k=1}^N \sum_{i=1}^3 m_k \frac{\partial U_{i_k}}{\partial q_s} \frac{\partial U_{i_k}}{\partial q_r} \dot{q}_s \dot{q}_r \quad (\text{relative kinetic energy})$$

¹ $\mathcal{G}(q_1, \dots, q_n)$ is said to be a homogeneous function of degree p , where p is an integer, if $\forall \alpha \neq 0, \mathcal{G}(\alpha q_1, \dots, \alpha q_n) = \alpha^p \mathcal{G}(q_1, \dots, q_n)$



Hamilton's Principle for Conservative Systems and Lagrange Equations

Classification of Inertia Forces

- From Euler's theorem on homogeneous functions of degree p $\left(\sum_{s=1}^n q_s \frac{\partial \mathcal{G}(\mathbf{q})}{\partial q_s} = p \mathcal{G}(\mathbf{q}) \right)$ it follows that

$$\mathcal{T}_1 = \sum_{s=1}^n \dot{q}_s \frac{\partial \mathcal{T}_1}{\partial \dot{q}_s} \quad \text{and} \quad \mathcal{T}_2 = \frac{1}{2} \sum_{s=1}^n \dot{q}_s \frac{\partial \mathcal{T}_2}{\partial \dot{q}_s}$$

- Interpretation of the first two terms of the Lagrange equations

$$\begin{aligned} -\frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}}{\partial q_s} &= -\frac{d}{dt} \left(\frac{\partial \mathcal{T}_1}{\partial \dot{q}_s} + \frac{\partial \mathcal{T}_2}{\partial \dot{q}_s} \right) + \frac{\partial}{\partial q_s} (\mathcal{T}_0 + \mathcal{T}_1 + \mathcal{T}_2) \\ &= -\frac{\partial}{\partial t} \left(\frac{\partial \mathcal{T}_1}{\partial \dot{q}_s} \right) - \sum_{r=1}^n \frac{\partial^2 \mathcal{T}_1}{\partial \dot{q}_s \partial q_r} \dot{q}_r - \frac{d}{dt} \left(\frac{\partial \mathcal{T}_2}{\partial \dot{q}_s} \right) + \frac{\partial}{\partial q_s} (\mathcal{T}_0 + \mathcal{T}_1 + \mathcal{T}_2) \\ &= \underbrace{\left(-\frac{\partial}{\partial t} \left(\frac{\partial \mathcal{T}_1}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}_0}{\partial q_s} \right)}_{\text{transport}} + \underbrace{\left(-\frac{d}{dt} \left(\frac{\partial \mathcal{T}_2}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}_2}{\partial q_s} \right)}_{\text{relative}} \\ &\quad + \underbrace{\left(-\sum_{r=1}^n \frac{\partial^2 \mathcal{T}_1}{\partial \dot{q}_s \partial q_r} \dot{q}_r + \frac{\partial \mathcal{T}_1}{\partial q_s} \right)}_{\text{complementary}} \end{aligned}$$



- Hamilton's Principle for Conservative Systems and Lagrange Equations

- Classification of Inertia Forces

- The transport inertia forces are those obtained by setting $\dot{q}_r = 0$

$$\implies -\frac{\partial}{\partial t} \left(\frac{\partial \mathcal{T}_1}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}_0}{\partial q_s}$$

- The relative inertia forces are those obtained by assuming $\frac{\partial U_{ik}}{\partial t} = 0$

$$\implies -\frac{d}{dt} \left(\frac{\partial \mathcal{T}_2}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}_2}{\partial q_s}$$

- The complementary inertia forces are given by the remainder

$$\implies F_s = -\sum_{r=1}^n \frac{\partial^2 \mathcal{T}_1}{\partial \dot{q}_s \partial q_r} \dot{q}_r + \frac{\partial \mathcal{T}_1}{\partial q_s}$$



Hamilton's Principle for Conservative Systems and Lagrange Equations

Classification of Inertia Forces

- Note that since $T_1 = \sum_{r=1}^n \dot{q}_r \frac{\partial T_1}{\partial \dot{q}_r}$

$$\begin{aligned} F_s &= -\sum_{r=1}^n \frac{\partial^2 T_1}{\partial \dot{q}_s \partial q_r} \dot{q}_r + \frac{\partial T_1}{\partial q_s} = -\sum_{r=1}^n \frac{\partial^2 T_1}{\partial \dot{q}_s \partial q_r} \dot{q}_r + \sum_{r=1}^n \frac{\partial^2 T_1}{\partial q_s \partial \dot{q}_r} \dot{q}_r \\ &= \sum_{r=1}^n \left(\frac{\partial^2 T_1}{\partial q_s \partial \dot{q}_r} - \frac{\partial^2 T_1}{\partial q_r \partial \dot{q}_s} \right) \dot{q}_r = \sum_{r=1}^n g_{rs} \dot{q}_r \end{aligned}$$

where the coefficients $g_{rs} = -g_{sr}$ do not depend on the velocities \dot{q}_s , but only on the generalized displacements and time

- The complementary inertia forces have the nature of Coriolis or gyroscopic forces: because of the skew-symmetry of the coefficients g_{rs} , it follows that

$$\sum_{s=1}^n F_s \dot{q}_s = \sum_{s=1}^n \sum_{r=1}^n g_{rs} \dot{q}_r \dot{q}_s = 0$$



└ Hamilton's Principle for Conservative Systems and Lagrange Equations

└ Classification of Inertia Forces

- Leonhard Euler (15 April 1707 - 18 September 1783)



- Swiss mathematician and physicist
- contributions: infinitesimal calculus and graph theory
- impact: mechanics, fluid dynamics, optics, and astronomy
- the asteroid 2002 Euler was named in his honor



└ Hamilton's Principle for Conservative Systems and Lagrange Equations

└ Energy Conservation in a System with Scleronomic Constraints

- Assume here, **until otherwise stated**, that the generalized displacements U_{ik} are independent explicitly of time $\Rightarrow \mathcal{T}_0 = \mathcal{T}_1 = 0$

$$\mathcal{T} = \mathcal{T}_2 = \frac{1}{2} \sum_{s=1}^n \sum_{r=1}^n \sum_{k=1}^N \sum_{i=1}^3 m_k \frac{\partial U_{ik}}{\partial q_s} \frac{\partial U_{ik}}{\partial q_r} \dot{q}_s \dot{q}_r = \frac{1}{2} \sum_{s=1}^n \dot{q}_s \frac{\partial \mathcal{T}_2}{\partial \dot{q}_s}$$

- Differentiation with respect to time of the above expression leads to

$$2 \frac{d\mathcal{T}}{dt} = \sum_{s=1}^n \ddot{q}_s \frac{\partial \mathcal{T}}{\partial \dot{q}_s} + \sum_{s=1}^n \dot{q}_s \frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) \quad (3)$$

- From $\mathcal{T} = \mathcal{T}(\mathbf{q}, \dot{\mathbf{q}})$ it follows that

$$\frac{d\mathcal{T}}{dt} = \sum_{s=1}^n \ddot{q}_s \frac{\partial \mathcal{T}}{\partial \dot{q}_s} + \sum_{s=1}^n \dot{q}_s \frac{\partial \mathcal{T}}{\partial q_s} \quad (4)$$

- (3) - (4) $\Rightarrow \frac{d\mathcal{T}}{dt} = \sum_{s=1}^n \dot{q}_s \left[\frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) - \frac{\partial \mathcal{T}}{\partial q_s} \right]$



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Energy Conservation in a System with Scleronomic Constraints

$$\frac{dT}{dt} = \sum_{s=1}^n \dot{q}_s \left[\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_s} \right) - \frac{\partial T}{\partial q_s} \right]$$

- From the Lagrange equations of motion ($-\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_s} \right) + \frac{\partial T}{\partial q_s} + Q_s = 0$), it follows that

$$\frac{dT}{dt} = \sum_{s=1}^n Q_s \dot{q}_s$$

- For a system with scleronomic constraints, $\mathcal{V} = \mathcal{V}(\mathbf{q})$
- Since for conservative forces $Q_s = -\frac{\partial \mathcal{V}}{\partial q_s}$, it follows that

$$\frac{dT}{dt} = \sum_{s=1}^n Q_s \dot{q}_s = -\sum_{s=1}^n \frac{\partial \mathcal{V}}{\partial q_s} \dot{q}_s = -\frac{d\mathcal{V}}{dt}$$

and therefore

$$\frac{d}{dt}(\mathcal{T} + \mathcal{V}) = 0 \Leftrightarrow \mathcal{E} = \mathcal{T} + \mathcal{V} = cst$$



└ Hamilton's Principle for Conservative Systems and Lagrange Equations

└ Classification of Generalized Forces

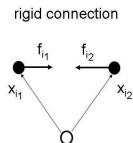
- Internal forces
 - linking forces
 - elastic forces
 - dissipation forces (may have external origins too)
- External forces
 - conservative forces
 - non-conservative forces
- Both types of forces are said to be conservative if the associated virtual work is recoverable



- Hamilton's Principle for Conservative Systems and Lagrange Equations

- Classification of Generalized Forces

- Linking forces



$$f_{i1} + f_{i2} = 0, \quad i = 1, \dots, 3$$

- virtual work: $\delta W = \sum_{i=1}^3 f_{i1} \delta u_{i1} + f_{i2} \delta u_{i2} = \sum_{i=1}^3 f_{i1} (\delta u_{i1} - \delta u_{i2})$
- for admissible virtual displacements – that is, virtual displacements that are compatible with the real displacements –
 $\delta u_{i1} = \delta u_{i2} \Rightarrow \delta W = 0$

Linking forces do NOT contribute to the generalized forces acting on the global system

- the above result is a nice aspect of Lagrangian mechanics



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- Elastic forces

- produced work is storable in a recoverable form
- internal energy $\mathcal{V}_{int}(u_{ik})$

- $$\delta\mathcal{T} = -\delta\mathcal{V}_{int} = -\sum_{k=1}^N \sum_{i=1}^3 \frac{\partial\mathcal{V}_{int}}{\partial u_{ik}} \delta u_{ik}$$

- $$\mathcal{V}_{int} = \mathcal{V}_{int}(\mathbf{q}) \Rightarrow \delta\mathcal{T} = -\delta\mathcal{V}_{int} = -\sum_{s=1}^n \frac{\partial\mathcal{V}_{int}}{\partial q_s} \delta q_s = \sum_{s=1}^n Q_s \delta q_s$$

$$\Rightarrow \boxed{Q_s = -\frac{\partial\mathcal{V}_{int}}{\partial q_s}}$$



Hamilton's Principle for Conservative Systems and Lagrange Equations

Classification of Generalized Forces

■ Dissipative forces

- remain parallel in opposite direction to the velocity vector \mathbf{v} and are function of its

$$\text{modulus } v_k = \sqrt{\sum_{i=1}^3 \dot{u}_{ik}^2}$$

$$\mathbf{f}_k = -C_k f_k(v_k) \frac{\mathbf{v}_k}{v_k}$$

■ virtual work

$$\mathbf{u} = \mathbf{u}(\mathbf{q}, t) \Rightarrow \delta W = \sum_{k=1}^N \sum_{i=1}^3 f_{ik} \delta u_{ik} = - \sum_{s=1}^n \sum_{k=1}^N \sum_{i=1}^3 C_k f_k(v_k) \frac{v_{ik}}{v_k} \frac{\partial u_{ik}}{\partial q_s} \delta q_s = \sum_{s=1}^n Q_s \delta q_s$$

$$\Rightarrow Q_s = - \sum_{k=1}^N \sum_{i=1}^3 C_k f_k(v_k) \frac{v_{ik}}{v_k} \frac{\partial u_{ik}}{\partial q_s}$$

- since $v_{ik} = \frac{du_{ik}}{dt} = \frac{\partial u_{ik}}{\partial t} + \sum_{s=1}^n \frac{\partial u_{ik}}{\partial q_s} \dot{q}_s$ and therefore $\frac{\partial v_{ik}}{\partial \dot{q}_r} = \frac{\partial u_{ik}}{\partial q_r}$, it follows that

$$Q_s = - \sum_{k=1}^N \sum_{i=1}^3 C_k f_k(v_k) \frac{v_{ik}}{v_k} \frac{\partial v_{ik}}{\partial \dot{q}_s} = - \sum_{k=1}^N C_k \frac{f_k(v_k)}{v_k} \frac{\partial}{\partial \dot{q}_s} \left(\frac{1}{2} \sum_{i=1}^3 v_{ik}^2 \right)$$

$$\Rightarrow Q_s = - \sum_{k=1}^N C_k f_k(v_k) \frac{\partial v_k}{\partial \dot{q}_s}$$



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$$Q_s = - \sum_{k=1}^N C_k f_k(v_k) \frac{\partial v_k}{\partial \dot{q}_s}$$

- Dissipation function:

$$\mathcal{D} = \sum_{k=1}^N \int_0^{v_k(\dot{q})} C_k f_k(\gamma) d\gamma$$

- From Leibniz's integral rule

$$\frac{d}{dt} \left(\int_{a(t)}^{b(t)} f(x, t) dx \right) = \int_{a(t)}^{b(t)} \frac{\partial f}{\partial t} dx + f(b(t), t) \frac{db}{dt} - f(a(t), t) \frac{da}{dt}$$

it follows that

$$\frac{\partial \mathcal{D}}{\partial \dot{q}_s} = \sum_{k=1}^N C_k f_k(v_k) \frac{\partial v_k}{\partial \dot{q}_s} \Rightarrow Q_s = - \frac{\partial \mathcal{D}}{\partial \dot{q}_s}$$

- Dissipated power: $P = \sum_{s=1}^n Q_s \dot{q}_s = - \sum_{s=1}^n \dot{q}_s \frac{\partial \mathcal{D}}{\partial \dot{q}_s}$



└ Hamilton's Principle for Conservative Systems and Lagrange Equations

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- If \mathcal{D} is assumed to be a homogeneous function of order p in the generalized velocities, then

$$\frac{d\mathcal{T}}{dt} = \sum_{s=1}^n Q_s \dot{q}_s = -\frac{d\mathcal{V}}{dt} - \sum_{s=1}^n \dot{q}_s \frac{\partial \mathcal{D}}{\partial \dot{q}_s} = -\frac{d\mathcal{V}}{dt} - p\mathcal{D}$$

$$\implies \boxed{\frac{d}{dt}(\mathcal{T} + \mathcal{V}) = -p\mathcal{D}}$$

- $p = 1$: dry friction
- $p = 2$: viscous damping
- $p = 3$: aerodynamic drag



Hamilton's Principle for Conservative Systems and Lagrange Equations

Classification of Generalized Forces

- Conservative external forces: these derive from a potential

$$\Rightarrow \exists \mathcal{V}_{\text{ext}} / Q_s = -\frac{\partial \mathcal{V}_{\text{ext}}}{\partial q_s}$$

\Rightarrow their virtual work during a cycle is zero

$$\delta W = \oint Q_s \delta q_s = - \oint \frac{\partial \mathcal{V}_{\text{ext}}}{\partial q_s} \delta q_s = - \oint \delta \mathcal{V}_{\text{ext}} = 0$$

- Non-conservative external forces

$$\delta W = \sum_{s=1}^n Q_s \delta q_s = \sum_{k=1}^N \sum_{i=1}^3 f_{ik} \delta u_{ik} = \sum_{i=1}^3 \sum_{k=1}^N \sum_{s=1}^n f_{ik} \frac{\partial u_{ik}}{\partial q_s} \delta q_s$$

$$\Rightarrow Q_s = \sum_{k=1}^N \sum_{i=1}^3 f_{ik} \frac{\partial u_{ik}}{\partial q_s}$$



└ Hamilton's Principle for Conservative Systems and Lagrange Equations

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- Summary: taking into account the non-conservative external forces, the power balance of a system solicited by internal and external forces can be written as

$$\frac{d}{dt}(\mathcal{T} + \mathcal{V}) = -p\mathcal{D} + \sum_{s=1}^n Q_s \dot{q}_s$$



Lagrange Equations in the General Case

- In the general case of a non-conservative system with rheonomic constraints, the Lagrange equations of motion can be written as

$$-\frac{d}{dt} \left(\frac{\partial \mathcal{T}}{\partial \dot{q}_s} \right) + \frac{\partial \mathcal{T}}{\partial q_s} - \frac{\partial \mathcal{V}}{\partial q_s} - \frac{\partial \mathcal{D}}{\partial \dot{q}_s} + Q_s(t) = 0$$

where

$$\begin{aligned} \mathcal{V} = \mathcal{V}_{ext} + \mathcal{V}_{int} &= \text{total potential} \\ \mathcal{D} &= \text{dissipation function} \\ Q_s(t) &= \text{non-conservative external generalized forces} \end{aligned}$$

- They can also be written in terms of the inertia forces as follows

$$\frac{d}{dt} \left(\frac{\partial \mathcal{T}_2}{\partial \dot{q}_s} \right) - \frac{\partial \mathcal{T}_2}{\partial q_s} = Q_s(t) - \frac{\partial \mathcal{V}^*}{\partial q_s} - \frac{\partial \mathcal{D}}{\partial \dot{q}_s} + F_s - \frac{\partial}{\partial t} \left(\frac{\partial \mathcal{T}_1}{\partial \dot{q}_s} \right)$$

where

$$\begin{aligned} \mathcal{V}^* = \mathcal{V} - \mathcal{T}_0 &= \text{potential modified by the tranport kinetic energy} \\ F_s = \sum_{r=1}^n g_{rs} \dot{q}_r &= \text{generalized gyroscopic forces} \end{aligned}$$

